

HITACHI

THE CHALLENGE

- Optimizing the design of fluid dynamic bearings (FDBs) that are so small and inaccessible that it is difficult to see or measure them
- Building a scaled up physical model to validate a computational fluid dynamics (CFD) model

THE STORY

Ferdinand Hendriks of Hitachi Global Storage Technologies (Hitachi GST) has achieved what is probably the first validated simulation of oil air interfaces (OAI) in a high-speed fluid dynamic bearing (FDB). FDBs, which have largely replaced ball bearings in hard disk drives (HDD), are so tiny that the OAI is extremely hard to observe or measure. Ferdinand Hendriks' simulations of the OAI have contributed to a much improved understanding of the behavior of FDB under extreme circumstances.

THE BENEFITS

- The CFD model demonstrated the ability to predict the performance of the scale-up model
- In the future, the CFD model can be scaled down and used to investigate alternative FDB designs

Hitachi Global Storage Technologies (Hitachi GST) was founded in 2003 as a result of the strategic combination of IBM and Hitachi's storage technology businesses.

Hitachi GST Successfully Models

Bearings of Hard Disk Drives

Oil-Air Interfaces in Fluid Dynamic

Hitachi GST is positioned to immediately advance the role of hard disk drives beyond traditional computing environments to consumer electronics and other emerging applications.

Hitachi GST offers a comprehensive product portfolio unsurpassed in the industry - including 1-inch, 1.8-inch, 2.5-inch, and 3.5-inch hard disk drive storage devices.

DIFFICULT DESIGN CHALLENGE

FDBs are preferred over ball bearings in hard disk drives because the elimination of metal-tometal contact greatly reduces non-repeatable runout due to surface imperfections. But ever since their introduction in the 1960s by Prof. E. Muijderman for use in ultra-centrifuges, it has been well known that FDBs provide a very difficult design challenge. A few milligrams of oil must perform flawlessly in the FDB over the life of the HDD. The oil must not degrade, evaporate or atomize without oil changes or an oil filter.



Courtesy: Hitachi

The bearing surfaces in an FDB consist of a rotating (rotor) and stationary part (stator) with a thin layer of oil in between. Slanted grooves in either the rotor or stator generate oil film pressure to the support the rotor. In FDB designs for high-speed server-class drives, the rotor usually moves around the shaft without contact, the shaft being rigidly supported at both ends. The oil-air interface (OAI) occurs in the partial fill design of FDB's when the oil covers the grooved areas only partially.

Controlling the OAI is often considered to be key to successful FDB design because gas/fluid interfaces strongly affect the rotordynamics. The OAIs must be stable, especially when located in the grooved section of the FDB, in order to avoid formation of bubbles. The challenge of analyzing FDBs is increased by the fact that FDBs are so small and inaccessible that the OAI cannot be easily observed or measured. For example, the radius of the shaft of the FDBs is a couple of millimeters and the clearance between rotor and stator is only a few microns.

HARDWARE SCALE-UP MODEL

Ferdinand Hendriks has made major progress towards overcoming this obstacle by building an FDB that is scaled up by a factor of 1000, see Fig. 1. Ferdinand Hendriks selected the 1000X scale-up factor in order to achieve a minimum scaled up bearing clearance of 3 mm, which is sufficient for measuring and observing. In order to reduce the size of bearing to a manageable level while keeping the 3 mm bearing clearance, he modified the design of the scaled-up hardware model so that it consists of a single inclined groove in the stator wrapped around itself, and a smooth cylindrical rotor.

CFD SIMULATION

Simultaneously, Ferdinand Hendriks used CFD-ACE+ from ESI Group to develop a CFD model of the scaled up bearing. The basic idea is that after validating the CFD model against the physical experiments, it should be possible to reduce the CFD model to the size of the actual FBD while maintaining its accuracy. Then Ferdinand Hendriks will be able to use the actual size simulation model to evaluate the effects of potential design changes and optimize FDB design.

The Volume of Fluid (VOF) model used to track the gas/fluid interface in CFD-ACE+ successfully identified all of the major features of the OAI dynamics. These include a difference in OAI deflection for the stator and rotor, unsteady flow in the grooves, fast revolution near the rotor and slow revolution near the stator.

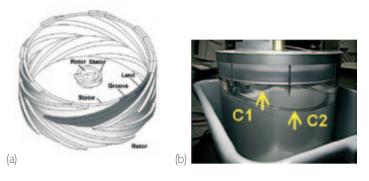


Figure 1. (a) Typical FDB herringbone bearing with 8 land/groove sections. To study a single land/groove pair shown in black, it was rolled up onto itself to form the tiny object at the center.

The final model (b) is obtained after scaling up by a factor 1000. It is shown filled with silicone 350 cS oil and running at scale speed. The scaled up rotation time is approximately 7 seconds. C1 is the oil contact line on the inner, rotating cylinder, C2 is the deformed contact line on the outer grooved stator.

The two-cell model correctly predicts that the oil contact line C1 at the rotor is not drawn into the groove but continues its circumferential movement while adhering to the rotor. However, the two-cell computations predicted non-steady flow while steady flow is observed in the experiment. It was recently found that a slight increase in the wetting angle can suppress the unsteadiness.

In the near future, Ferdinand Hendriks plans to use the model to incorporate complications such as modeling the effect of dissolved gases on cavitation behavior in FDBs, a common occurrence in many lubrication situations

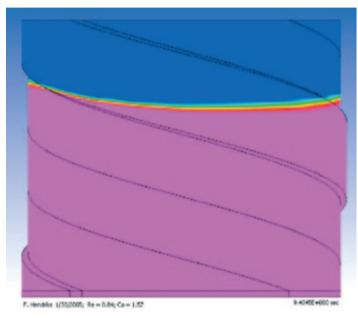


Figure 2. Computed average deflection of the oil air interface in the 1000X scale up model under 1G gravity. 1 cell discretization across the oil film. The average interface deflection agrees well with the deflection of the scale up experiment.

ABOUT ESI GROUP

Copyright © ESI Group, 2009 - G/RO/09.85/#

ESI is a world-leading supplier and pioneer of digital simulation software for prototyping and manufacturing processes that take into account the physics of materials. ESI has developed an extensive suite of coherent, industry-oriented applications to realistically simulate a product's behavior during testing, to fine-tune manufacturing processes in accordance with desired product performance, and to evaluate the environment's impact on product performance. ESI's products represent a unique collaborative and open environment for Simulation-Based Design, enabling virtual prototypes to be improved in a continuous and collaborative manner while eliminating the need for physical prototypes during product development. The company employs over 750 high-level specialists worldwide covering more than 30 countries. ESI Group is listed in compartment C of NYSE Euronext Paris. For further information, visit www.esi-group.com.

e i	BENELUX & SCANDINAVIA ESI Group Netherlands Radex Innovation Centre room 4.57	CZECH REPUBLIC & EASTERN EUROPE MECAS ESI Brojova 2113/16 326 00 Pilsen	FRANCE ESI France Parc d'Affaires Silic 99, rue des Solets - BP 80112	GERMANY ESI GmbH Mergenthalerallee 15-21 D-65760 Eschborn	ITALY ESI Italia Via San Donato 191 40127 Bologna Italy	SPAIN ESI Group Hispania Parque Empresarial Arroyo de la Vega C/ Francisca Deleado.	SWITZERLAND Calcom ESI Parc Scientifique EPFL / PSE-A 1015 Lausanne-EPFL	UNITED KINGDOM ESI UK 1 Robert: Robinson Av. The Magdalen Centre Oxford Science Park
nfo@esi-group.com	Rotterdamseweg 183 C 2629 HD Delft The Netherlands T. +31 (0)15 268 2501 F. +31 (0)15 268 2514	226 00 Pilsen Czech Republic T. +420 377 432 931 F. +420 377 432 930	80112 94513 Rungis cedex France T. +33 (0)1 49 78 28 00 F. +33 (0)1 46 87 72 02	Germany T. +49 (0)6196 9583 0 F. +49 (0)6196 9583 111	rtary T. +39 0516335577 T. +39 0516335578 F. +39 0516335601	C/ Francisca Delgado, 11 – planta 2ª 28108 Alcobendas (Madrid) Spain T. +34 91 484 02 56 F. +34 91 484 02 55	Switzerland T. +41 21 693 2918 F. +41 21 693 4740	Oxford OX 4 4GA Oxford OX 4 4GA United Kingdom T. +44 (0) 1865 784 830 F. +44 (0) 1865 784 826
SI Group	AMERICAS			ASIA				
teadquarters 20-102 Avenue de Suffren 5015 Paris RANCE + 33 (0)1 53 65 14 14 + 33 (0)1 53 65 14 12	USA ESI North America 32605 W 12 Mile Road Suite 350 Farmington Hills, MI 48334-3379 USA T. +1 (248) 381-8040 F. +1 (248) 381-8098	USA ESI Dorth America 6767 Old Madison Pike Suite 600 Huntsville, AL 35806 USA T. +1 (256) 713-4700 F. +1 (256) 713-4799	SOUTH AMERICA ESI South America AV. Pedroso de Morais, 16/9 c.j312 São Paulo SP CEP 05419-001 Brazil T./F.+55 (011) 3031-6221	CHINA ESI China Room 16A, Base F Fu Hua Mansion Nor.8 Chaoyangmen North Avenue Beijing 100027 China T. +86 (10) 6554 4907 F. +86 (10) 6554 4907	INDIA ESI India Indrakrupa #77,100 feet ring road 3rd phase, 6th block, Bangalore 560 085 India T. +91 98809 26626 F. +91 80401 74705	JAPAN ES Japan SF and 16 Shinjuku Green Tower Bidg, 6-14-1, Nichr-Shinjuku Shinjuku-ku, Tokyo 160-0023 Japan T, +81 3 6381 8490 / 8494 F, +81 3 6381 8488 / 8489	KOREA Hankook ESI 157-033, SF MISUNG bldg, 660-6, Deungchon-3Dong, Gangseo-ku, Seoul South Korea T, +82 2 3660 4500 F, +82 2 3662 0084	SOUTH-EAST ASIA ESI Group South-East Asi 12, Jalan Dato Haji Harun, Taman Taynton, Chensa 56000 Kuala Lumpur Malaysia T. +60 (12) 6181014